



# **Analysis and Evaluation of Fiberglass-Reinforced Plastic Pipe**

**Using CAESAR II**



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## **Agenda**



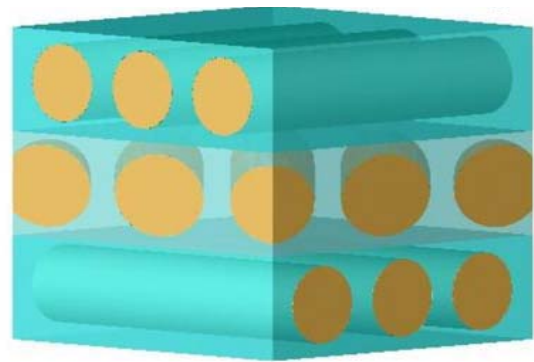
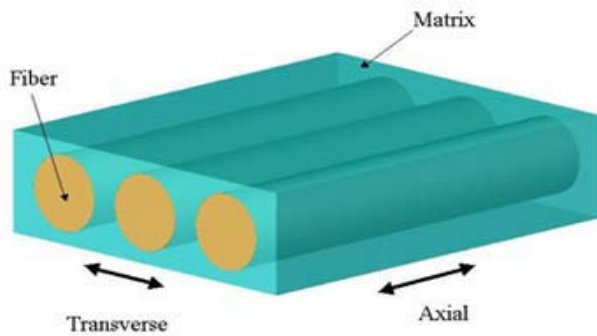
- **Fiberglass pipe**
- **Design using ISO 14692**
- **Using C2**



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# How do these materials together define the Mechanical properties?

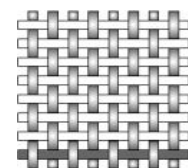


Laminate

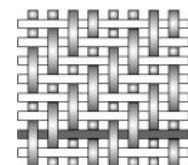
## Glass appearance in many ways



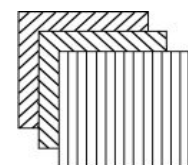
- Continuous roving
  - Direct
  - Assembled
- Chopped strand mats
- Continuous strand mats
- Woven fabrics
  - Unidirectional
  - Bi-directional
  - Multi-axial
- Knitted



Plain weave



twill weave



# Micro level analysis

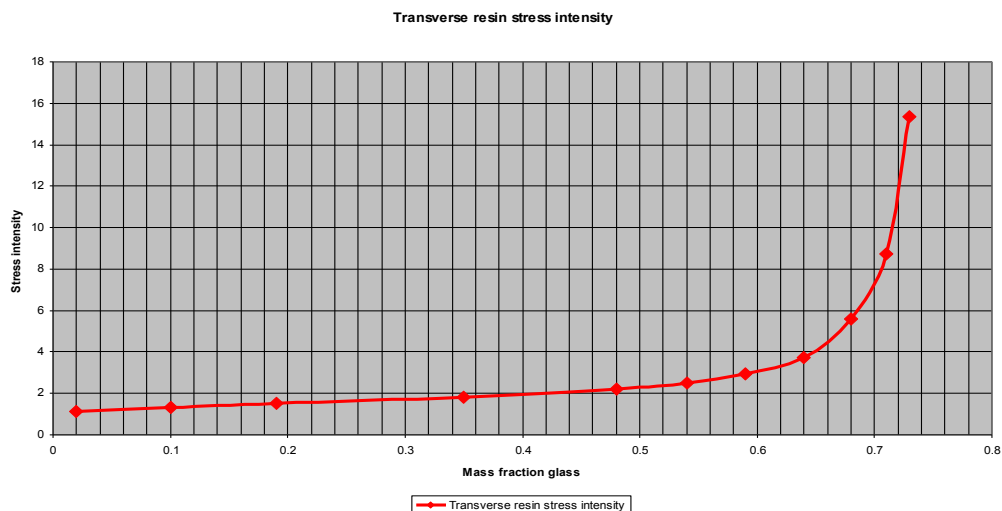


- Failure modes to be evaluated:
  - ☐ Failure of the fiber
  - ☐ Failure of the coupling agent layer
  - ☐ Failure of the matrix
  - ☐ Failure of the fiber-coupling agent bond
  - ☐ Failure of the coupling agent matrix bond
- Reduced failure mode evaluation (lack of detailed knowledge of 2, 4 and 5)
  - ☐ Fiber failure
  - ☐ Matrix failure
  - ☐ Fiber-matrix interface failure

# Stress level dependent on glass resin ratio



- Maximizing glass- resin ratio is not favorable for loads perpendicular to the glass
- For larger glass resin ratios resin bridges become the weak spot

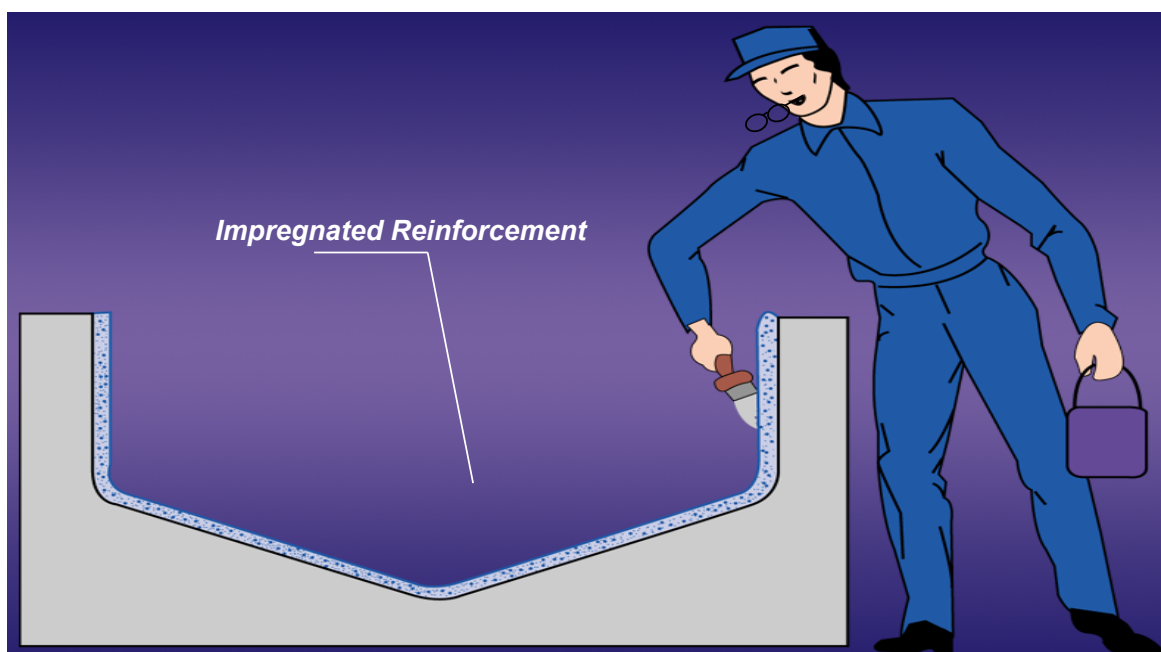


# Macro level analysis instead of Micro level



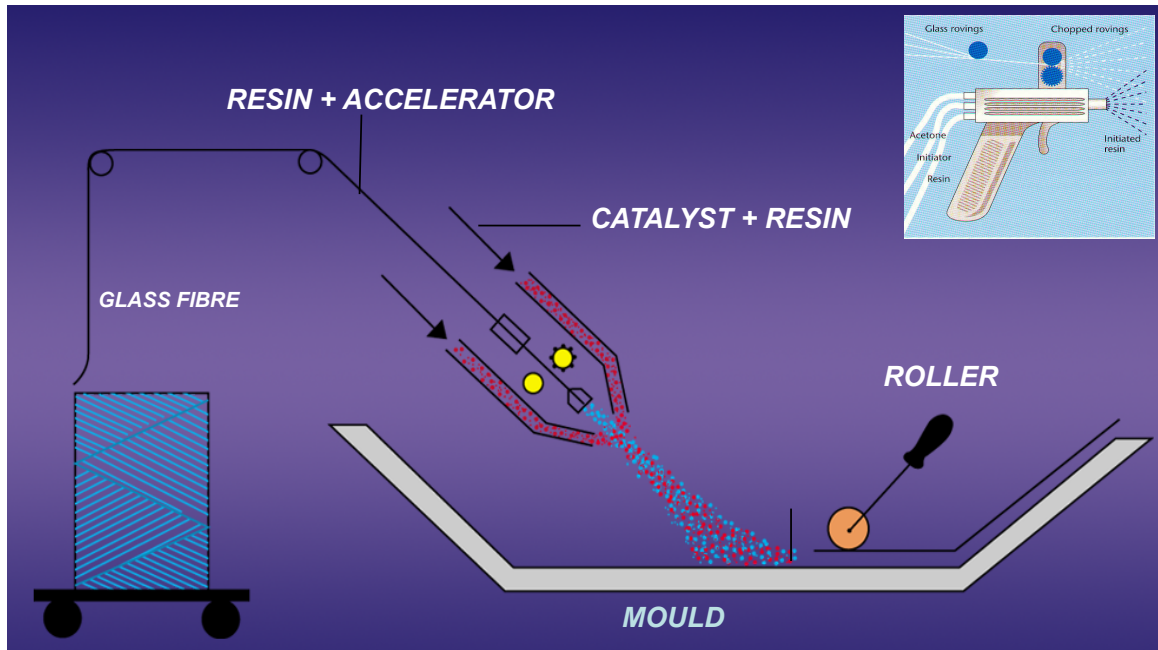
- **Micro level analysis**
  - Feasible in concept
  - Not feasible in practice since many fibers randomly distributed and oriented
- **Mini level analysis.**
  - Evaluation of individual laminate layers
  - Laminate layer is considered a continuum with material properties and failure modes
  - Assessment by averaging over cross-section.
- **Macro level analysis. (actual state of the art)**
  - Evaluation of components made from multiple laminate layers
  - Series of layers act as a homogeneous material with estimated properties based on layer properties and winding angle
  - Failure analysis based on equivalent stress

# Hand Lay-up processing technique

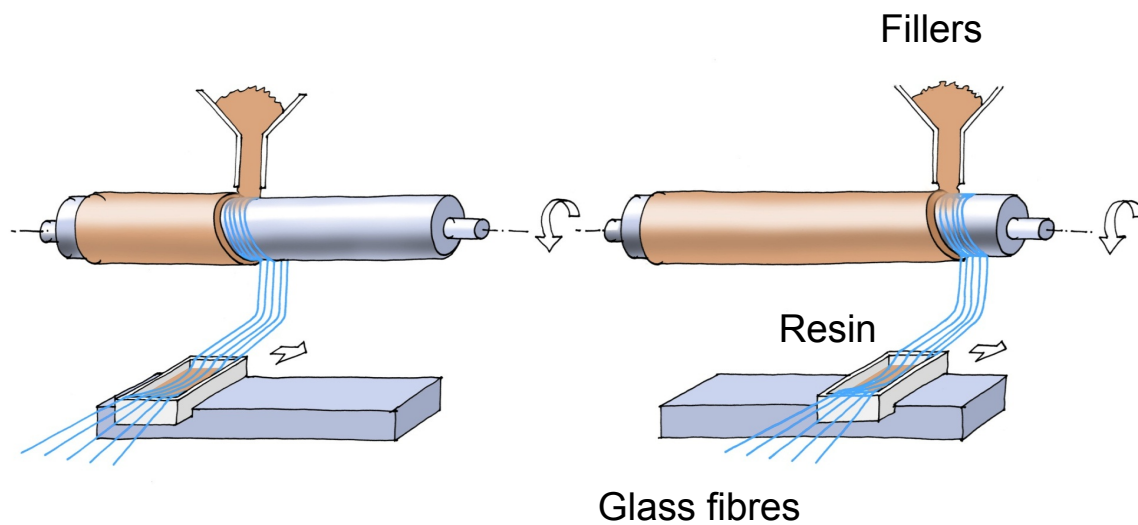




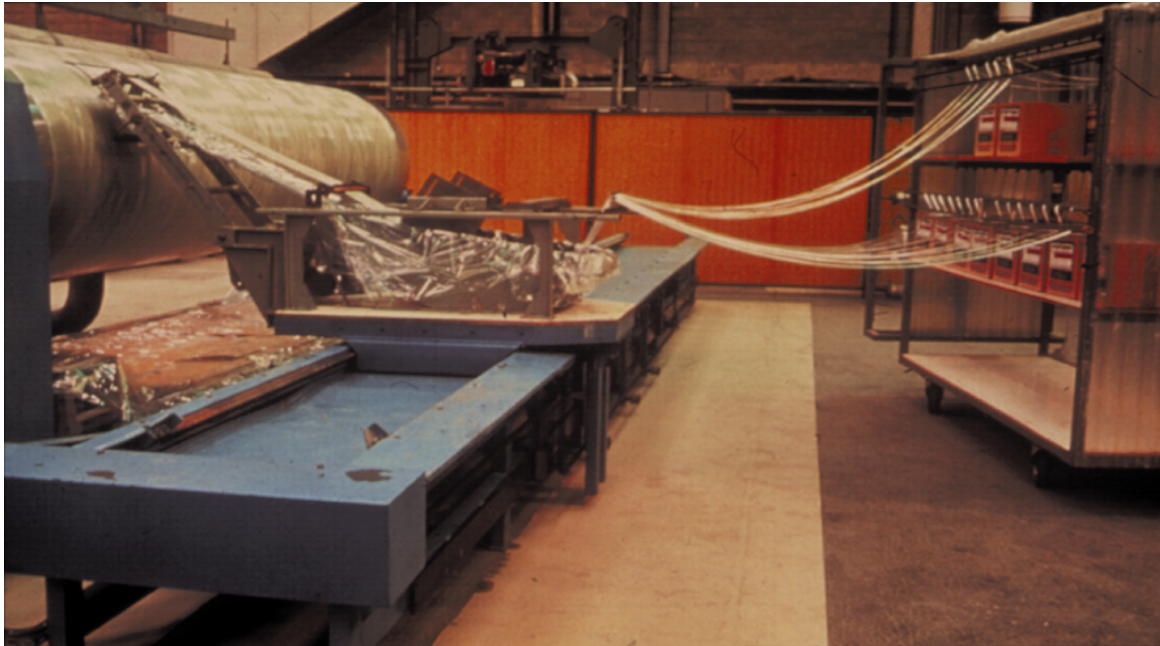
# Spray-up processing technique



# Filament winding processing technique



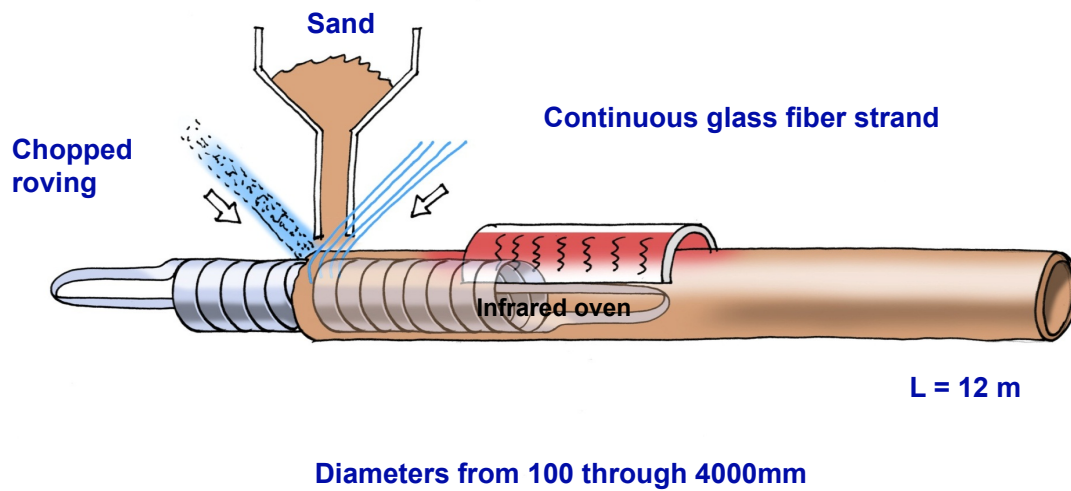
## Filament winding in action



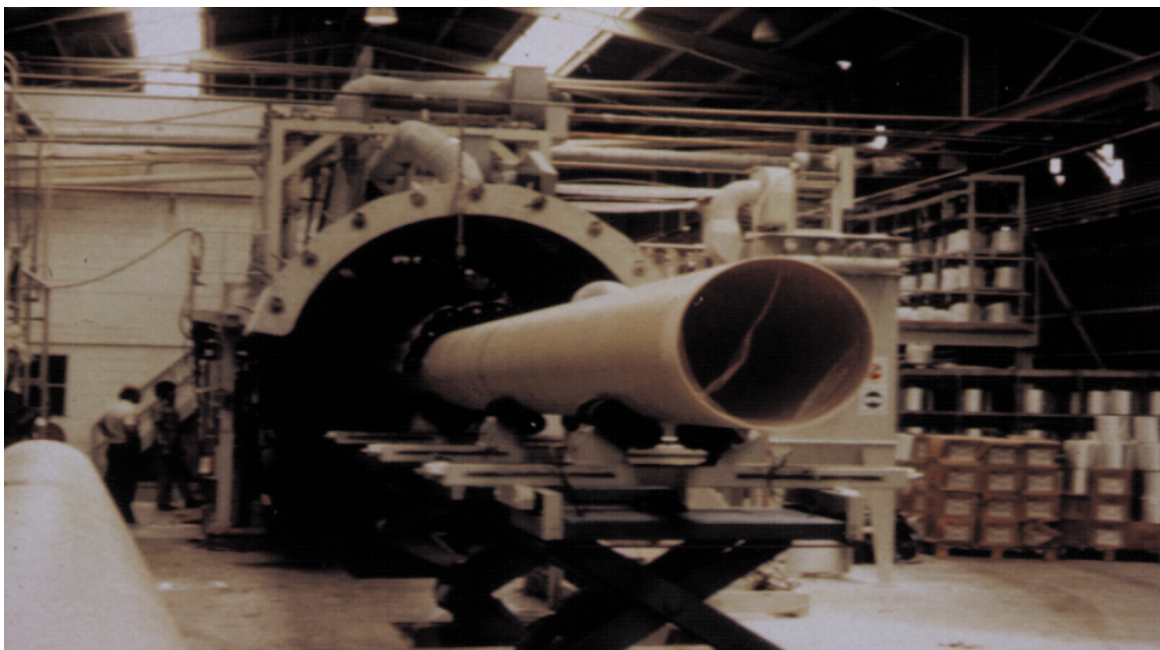
## Helical winding processing technique



# Continuous filament winding processing technique

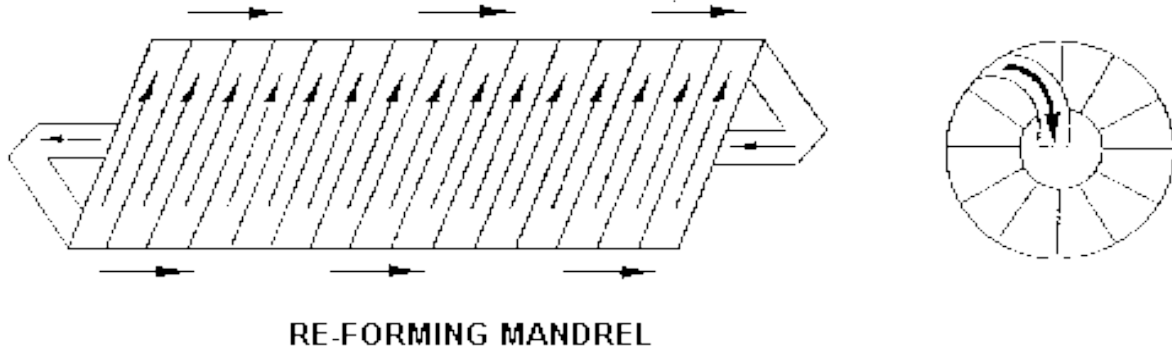


## Continuous filament winding in action





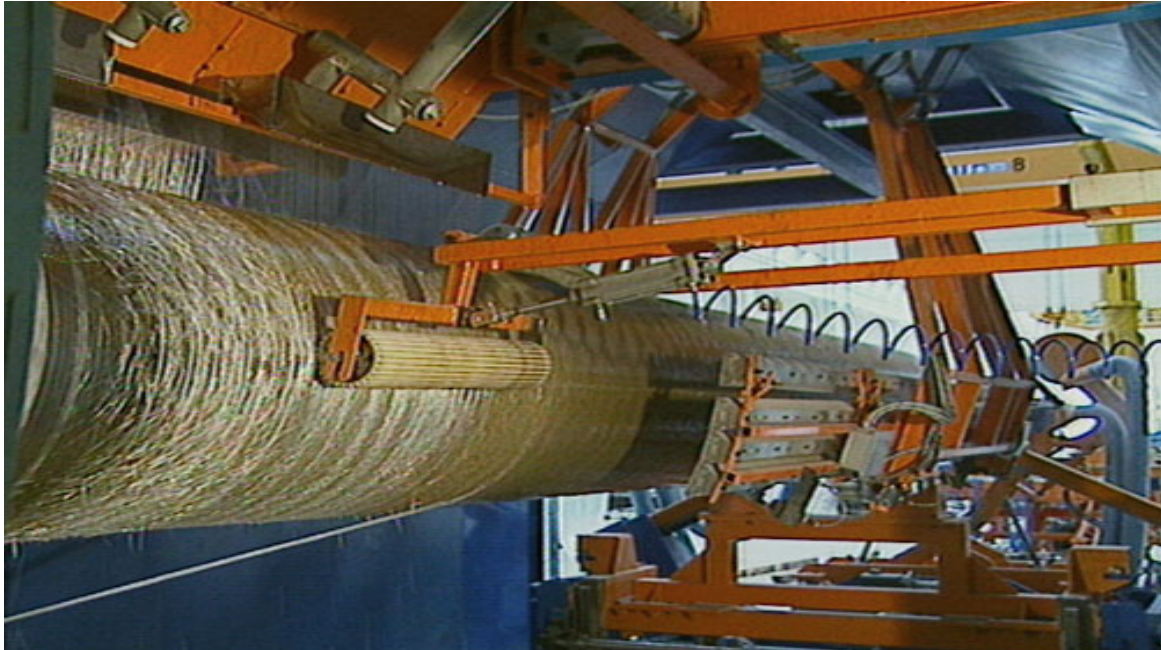
## Continuous filament winding in action



## Continuous filament winding in action



## Continuous filament winding in action

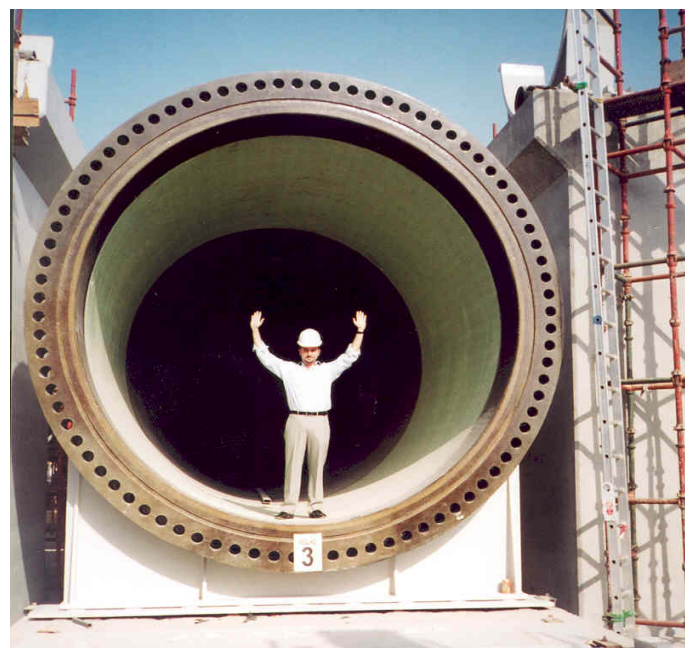


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## Before looking at the various design aspects first a little Fiberglass Quiz



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# How does typical specific thermal expansion coefficients compare?



	$\alpha$ [ $\mu\text{m}/\text{m}/^\circ\text{C}$ ]	
Stainless Steel	16.5	
Carbon Steel	11.0	
PVC	72.0	
Polyethelene	120.0	
Fiberglass	?	



	$\alpha = 50.0$ [ $\mu\text{m}/\text{m}/^\circ\text{C}$ ]
	$\alpha = 20.0$ [ $\mu\text{m}/\text{m}/^\circ\text{C}$ ]

# How does typical specific thermal expansion coefficients compare?



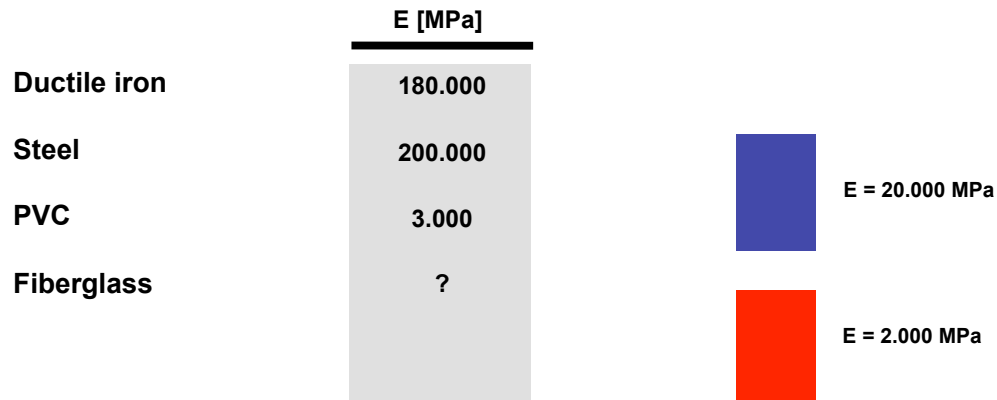
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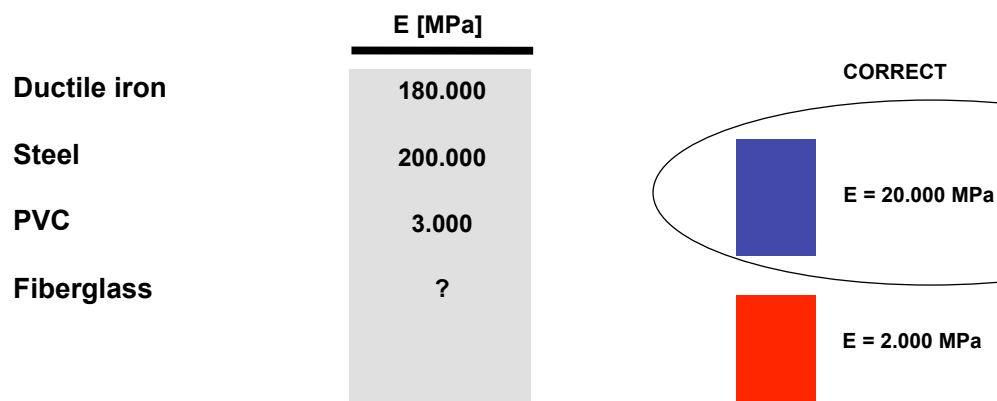
	$\alpha = 50.0$ [ $\mu\text{m}/\text{m}/^\circ\text{C}$ ]
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**CORRECT**

# How does the typical (volumetric) Elasticity of fiberglass compare?



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# What is the GRP required free bend leg length?



*Required free bend leg length for accommodation of the expansion.*

**Parameters:**

- ▶ Internal pressure: 20 Barg.
- ▶ Delta Temp.: 60 °C
- ▶ Expanding leg: 50 m

DIAMETER	GRP	STEEL
150mm	?	4m
200mm	?	4.6m



GRP: 5m and 5.5m



GRP: 7m and 8m

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**CORRECT**



# What is the typical wave speed in a Fiberglass pipe?



$$a = \sqrt{\frac{\frac{K}{\rho}}{1 + \frac{D}{t} \frac{K}{E}}}$$

K = fluid modulus of elasticity [Pa]

ρ = fluid density [kg/m³]

D = pipe diameter [mm]

t = pipe wall thickness [mm]

E = pipe modulus of elasticity [Pa]

Steel: 1000 - 1400m/s



GRP: 300 – 500m/s



GRP: 1300 – 1500m/s

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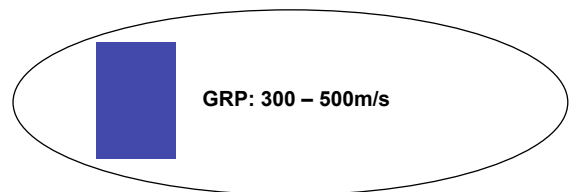
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CORRECT



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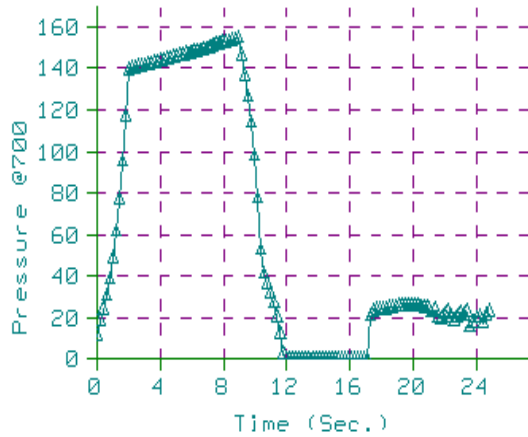


GRP: 1300 – 1500m/s

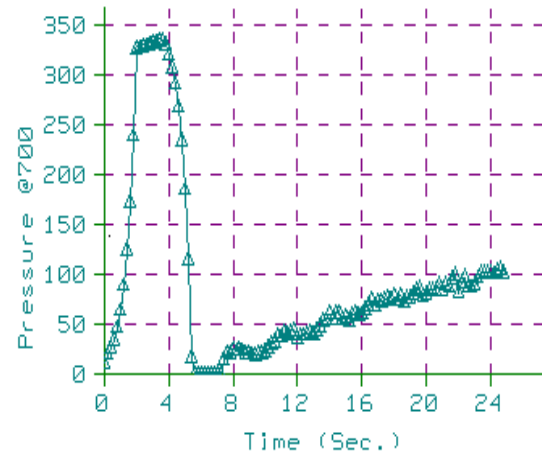
# Surge/Pressure Effects in GRP pipe in general are less than in metal pipe



## FIBERGLASS



## STEEL



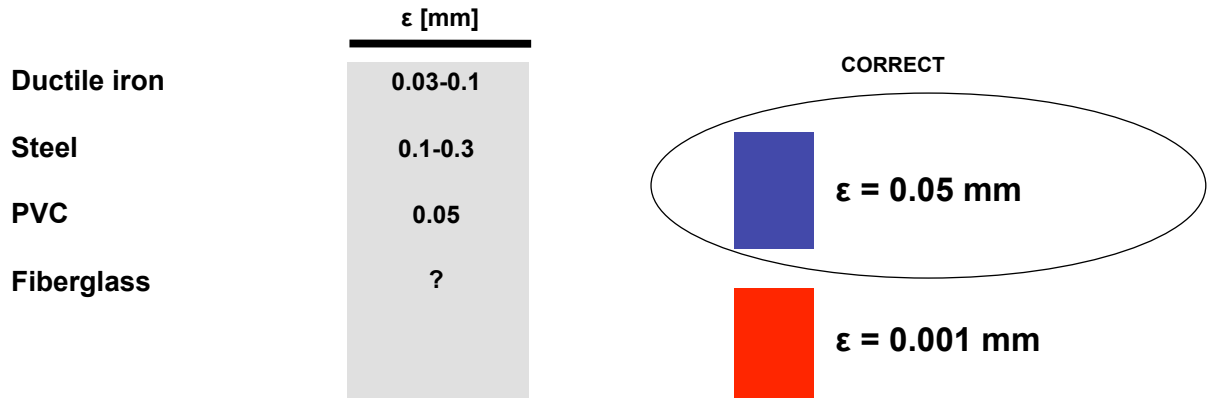
Pressure time history at valve (valve closure time: 2 secs)

# How does the effective material wall roughness of fiberglass compare?

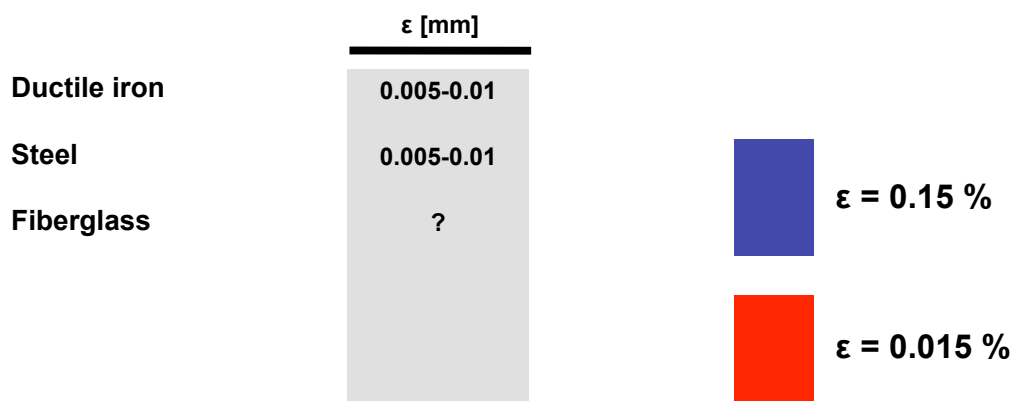


	$\epsilon$ [mm]	
Ductile iron	0.03-0.1	
Steel	0.1-0.3	
PVC	0.05	
Fiberglass	?	
		$\epsilon = 0.05$ mm
		$\epsilon = 0.001$ mm

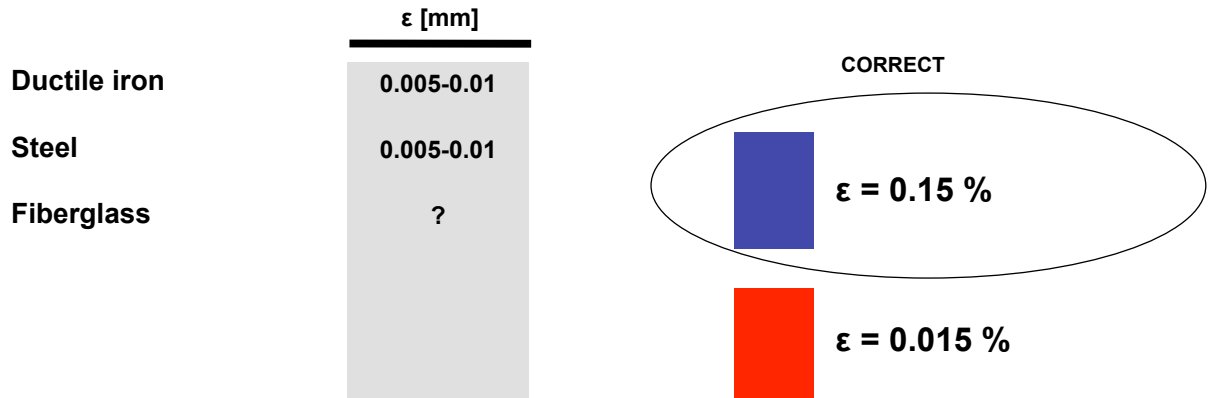
# How does the effective material wall roughness of fiberglass compare?



# How large is the effective material axial strain due to internal pressure ?



# How large is the typical material axial strain due to internal pressure ?



For  $\sigma_a = 18 \text{ MPa}$   $\epsilon_p = 18/12000 = 0.15\%$   
For  $\Delta T = 75^\circ\text{C}$   $\epsilon_T = 20 * 75 * 10^{-6} = 0.15 \%$

## Agenda



- Fiberglass pipe
- Design using ISO 14692
- Using C2

## ISO 14692 part 2 requires qualification of pipe and components



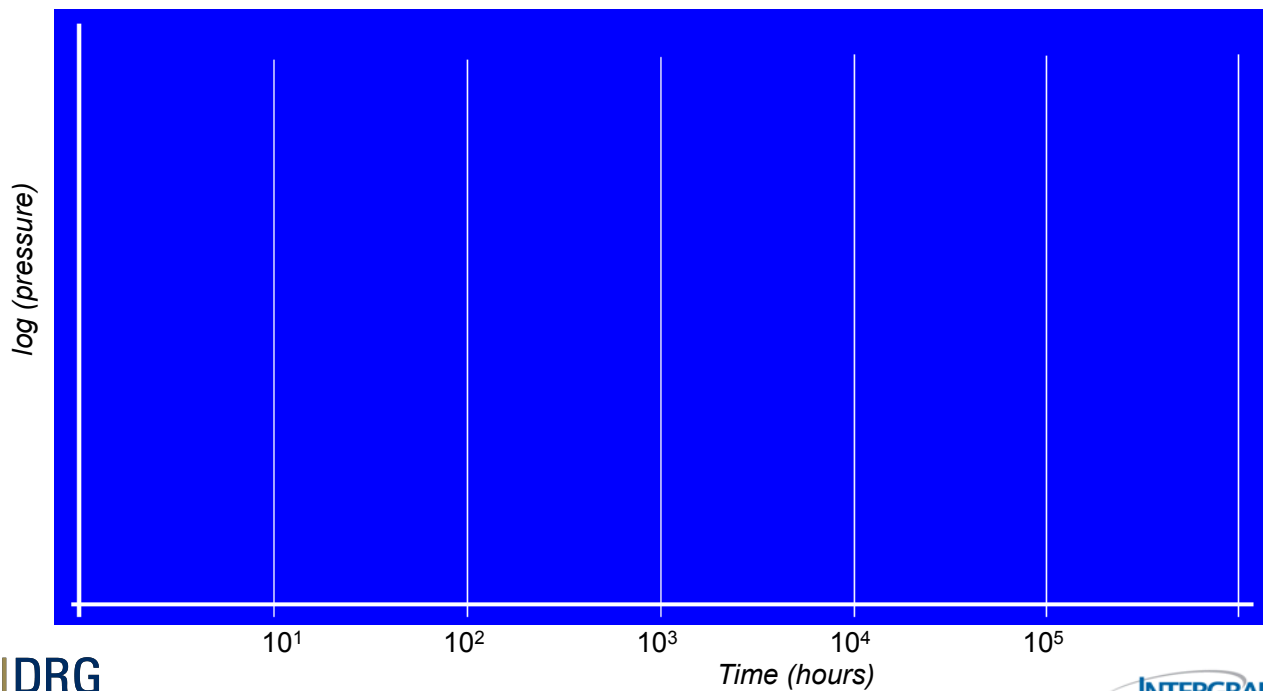
Small bore products (typically pipe):

- Long term regression test (ASTM D 2992)
- Delivers long-term strength
- Takes approximately two years

## Example of a burst test



## Qualification of GRP – ASTM D 2992

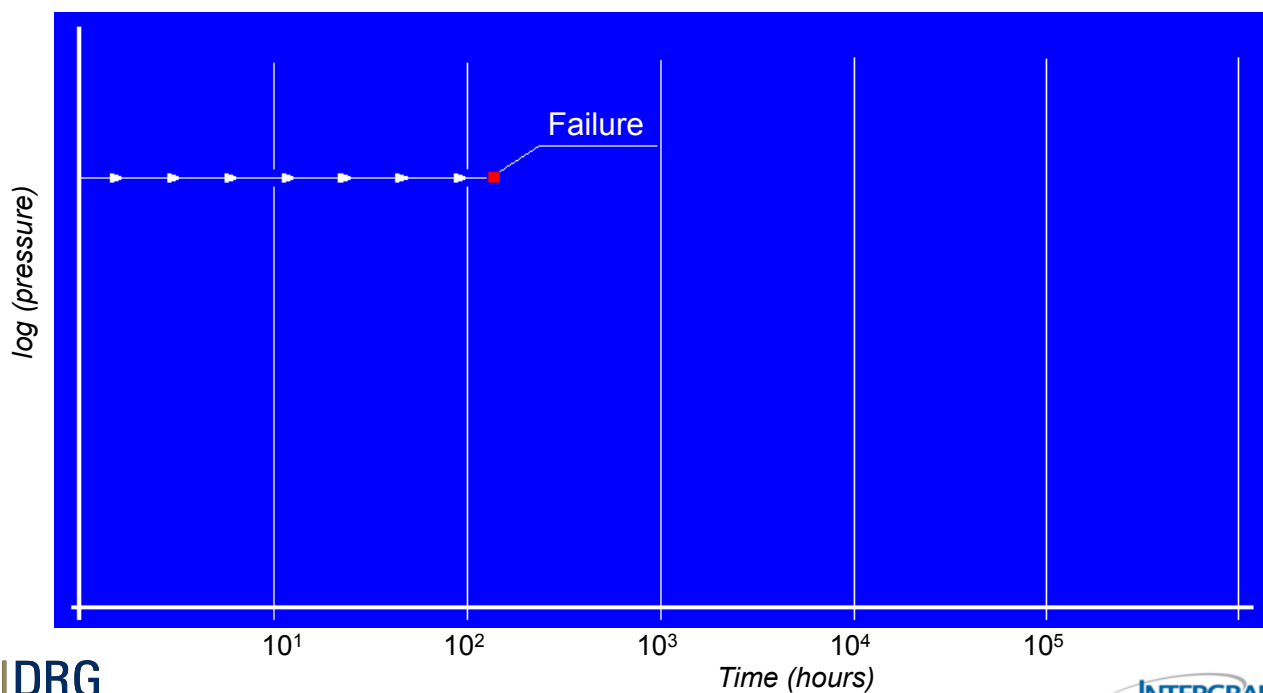


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## Qualification of GRP – ASTM D 2992

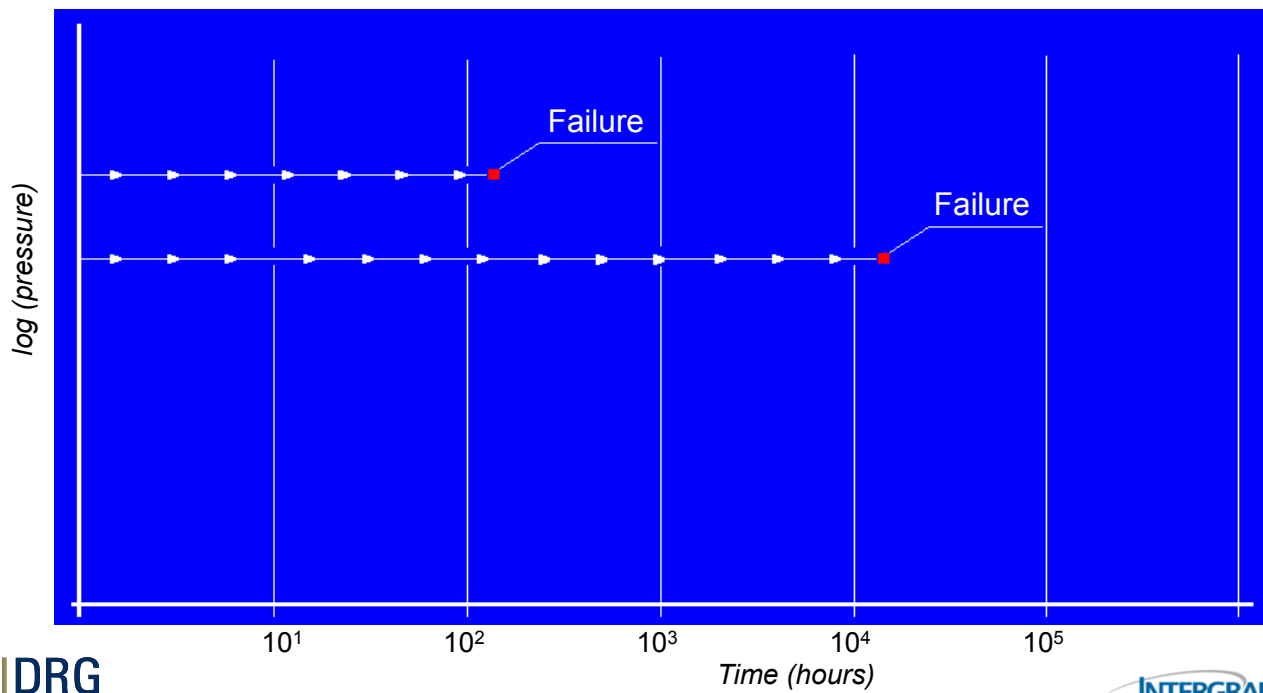


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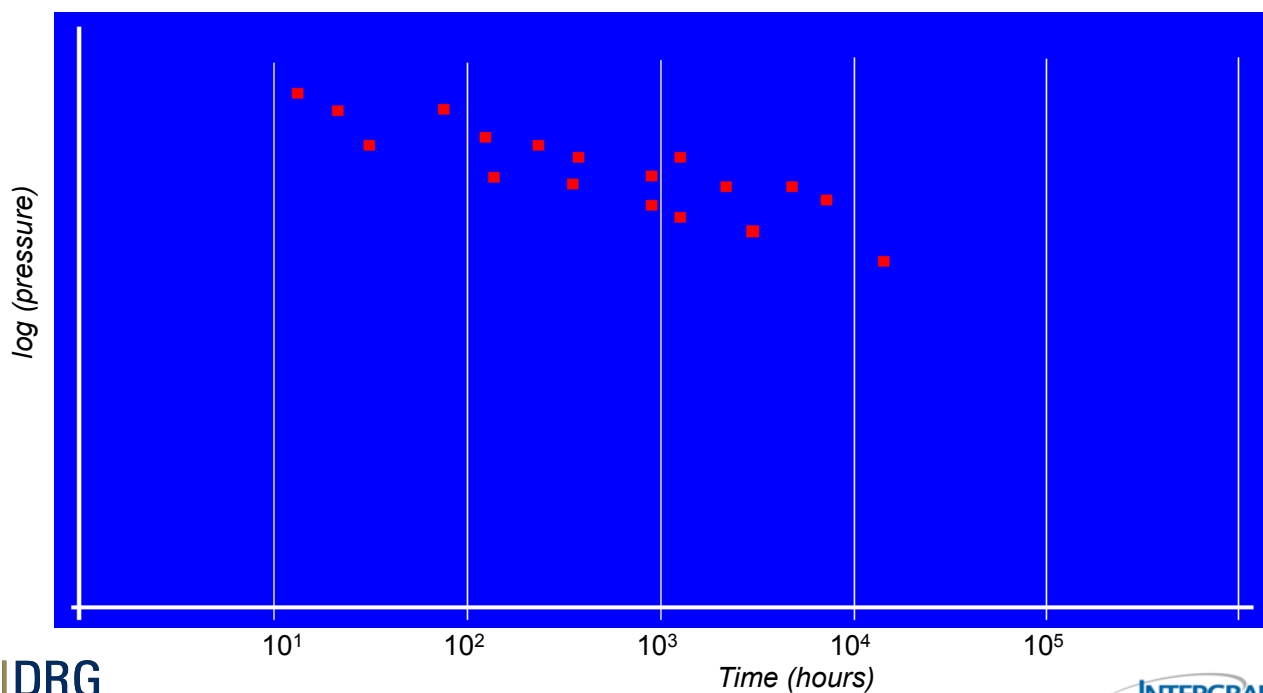
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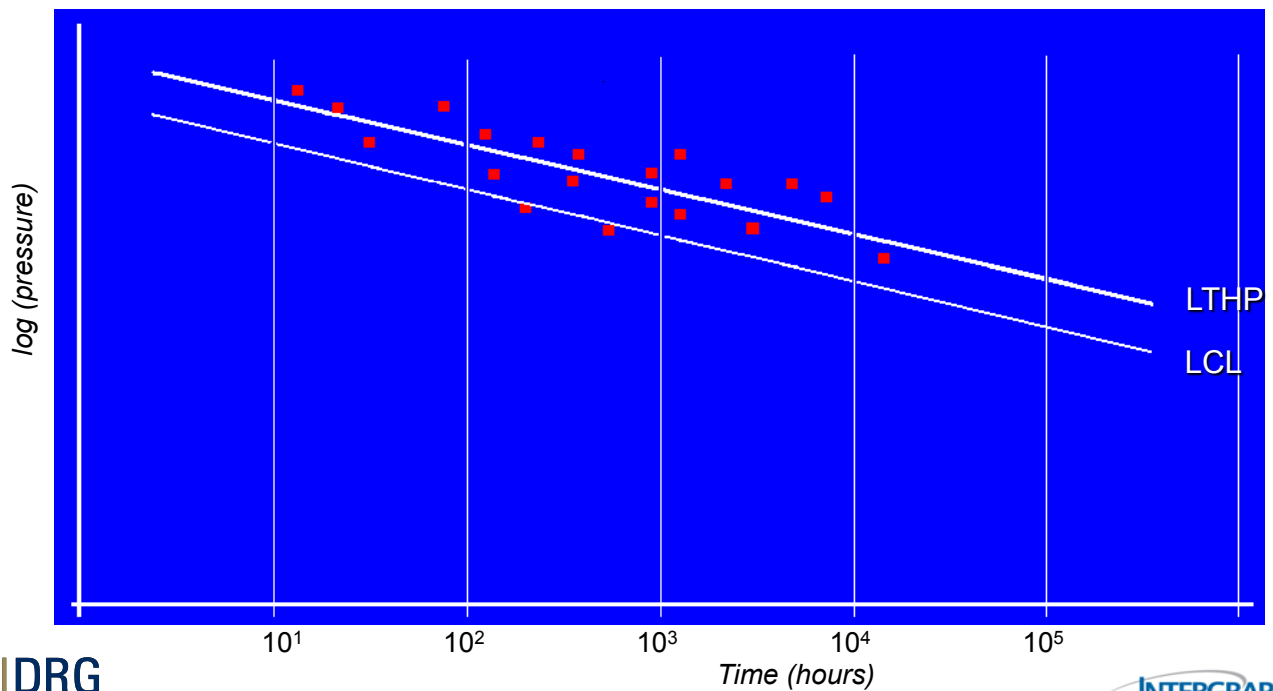
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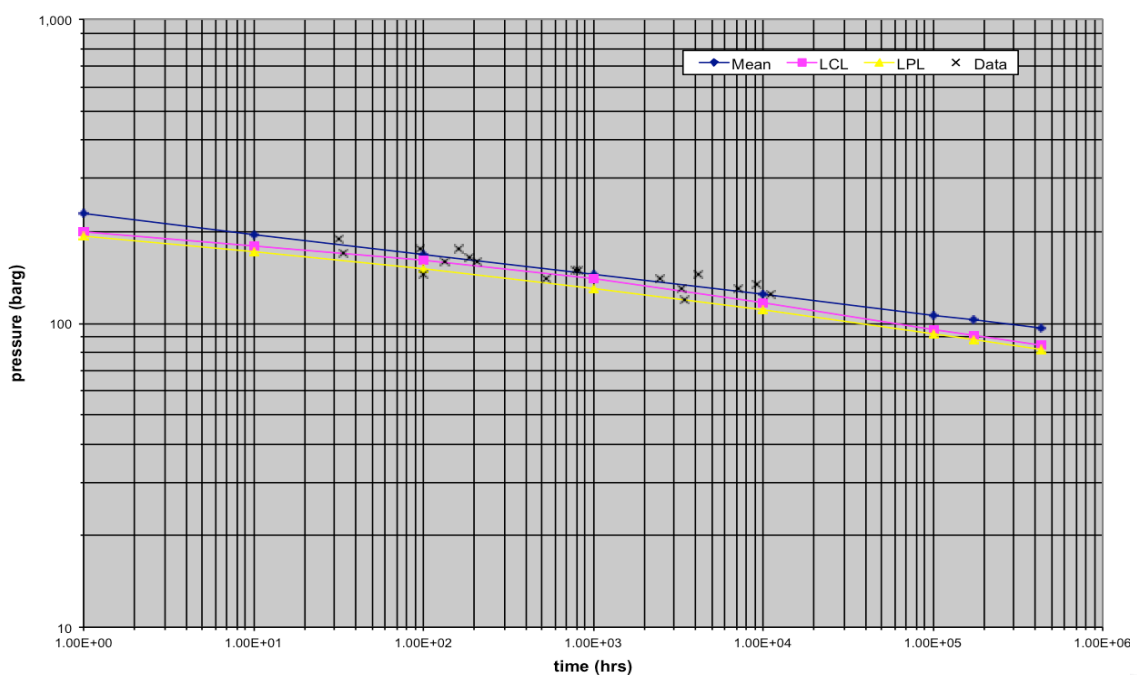
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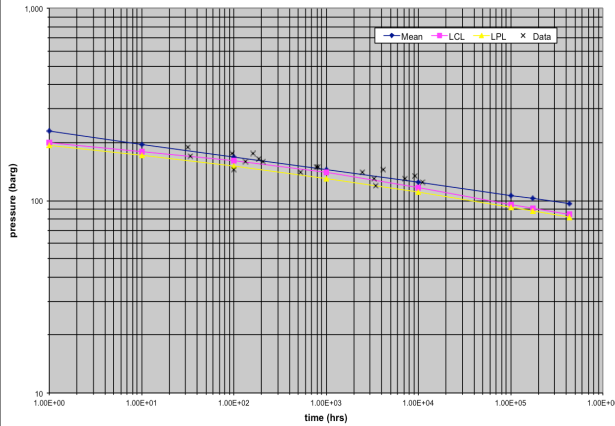




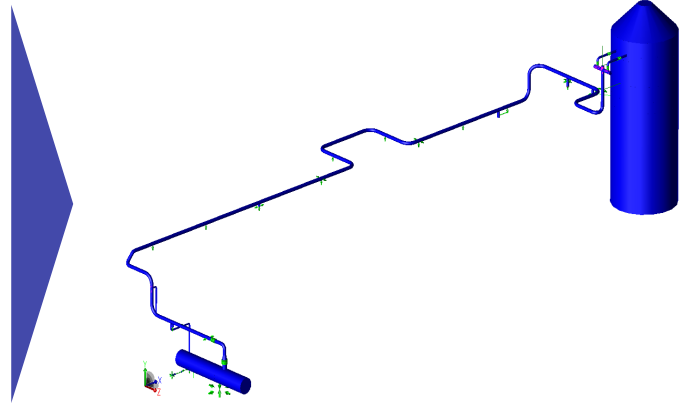
# Relation between long-term strength of piping and piping loads



## LONG-TERM STRENGTH



## PIPING LOADS



# GRP-pipe systems often fail due to poor or no engineering



- **Axial or hoop**
  - Hoop (5%) (Pipe system geometry is simple in circumferential direction)
  - Axial (95%) (Pipe system geometry is complex in axial direction)
- **When**
  - Small part (<5%) of the failures occurs during installation or operation
  - Most of the failures occur during hydro-testing (pressure testing)
- **Where**
  - Joints (89%)
  - Fittings (10%)
  - Plane pipe (1%)
- **Why**
  - Due to material defects (<2%)
  - Defective installation (49%)
  - Overloading of material due to shortcomings in design (49%)

## Axially Overloaded pipe failure

Also small bore can cause a lot of collateral damage



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## Axial tensile failure at reducer



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## Typical failures adjacent to bend



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## Failure in Pipe Adjacent to bend not in bend



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## Large tee failure during pressure testing



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## Axially Overloaded joint failure With large consequential damage due to waterhammer

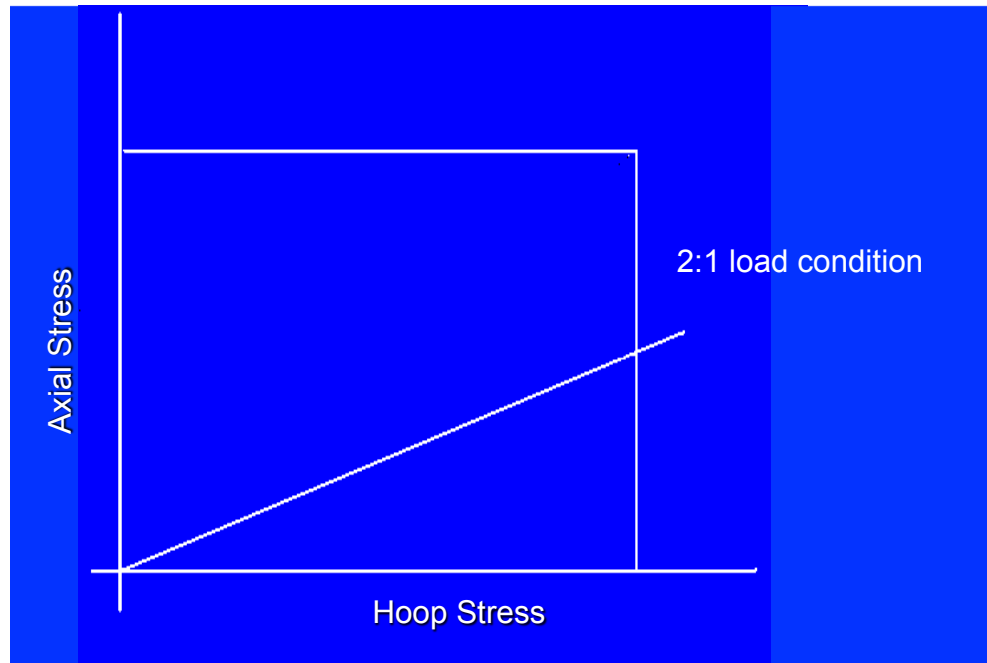


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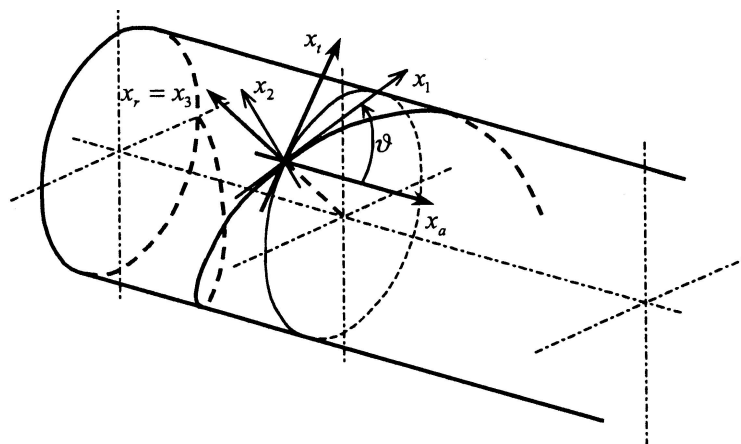
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# Construct a design envelope to assess pipe stresses against



## Effect of winding angle (netting theory)

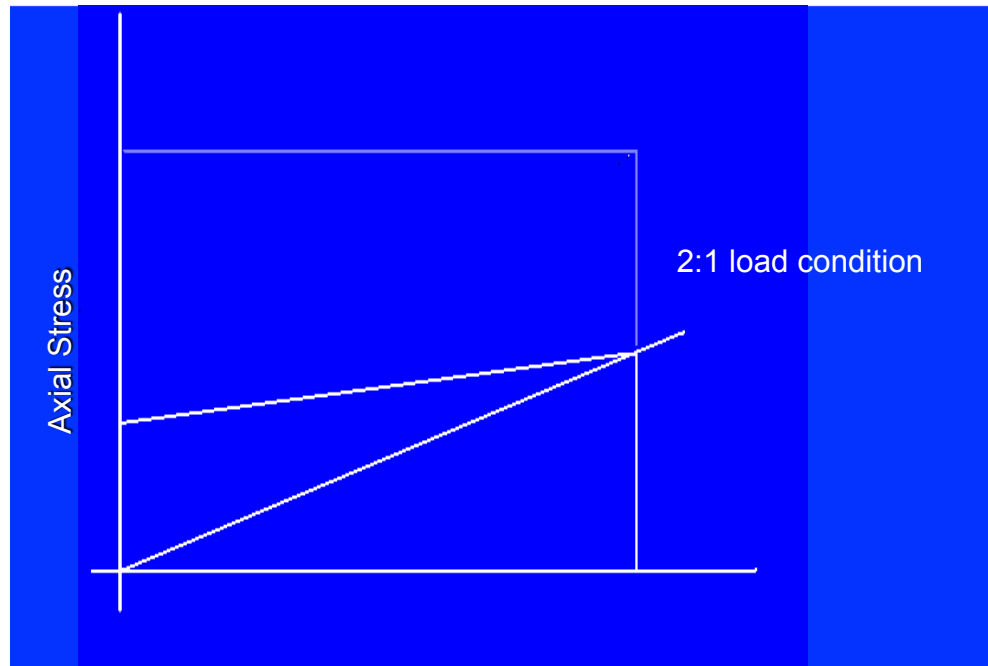


$$\Theta = 55^\circ: \sigma_{\text{hoop}}:\sigma_{\text{axial}} = 2:1$$

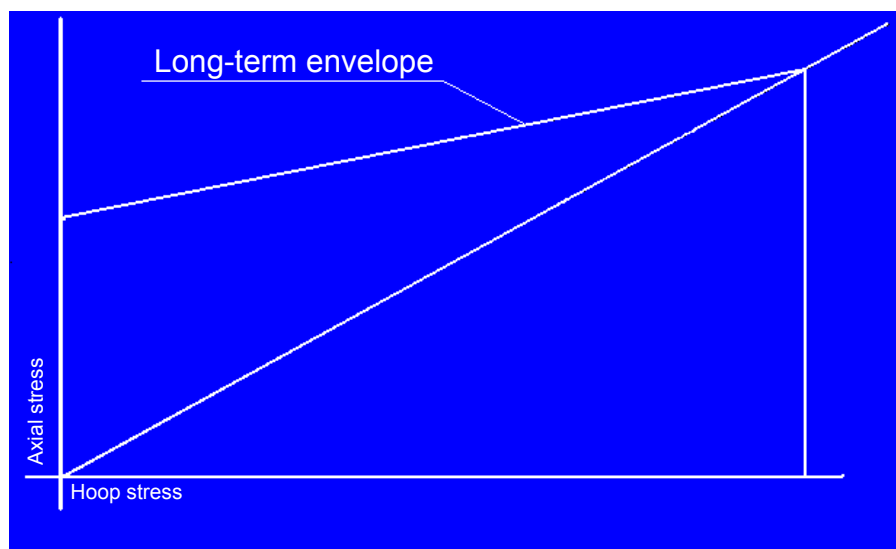
$$\Theta = 63^\circ: \sigma_{\text{hoop}}:\sigma_{\text{axial}} = 4:1$$

$$\Theta = 73^\circ: \sigma_{\text{hoop}}:\sigma_{\text{axial}} = 10:1$$

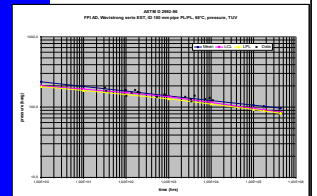
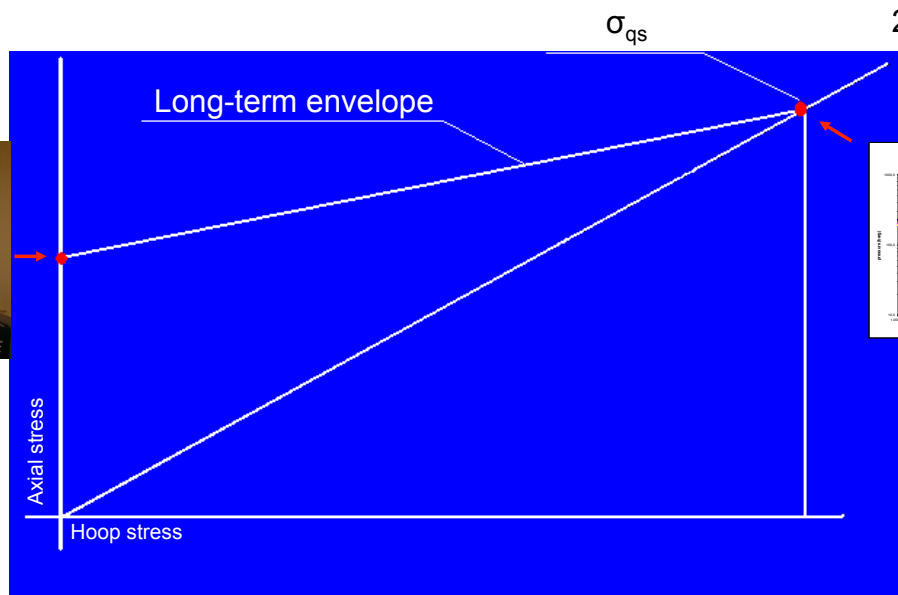
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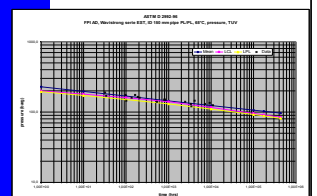
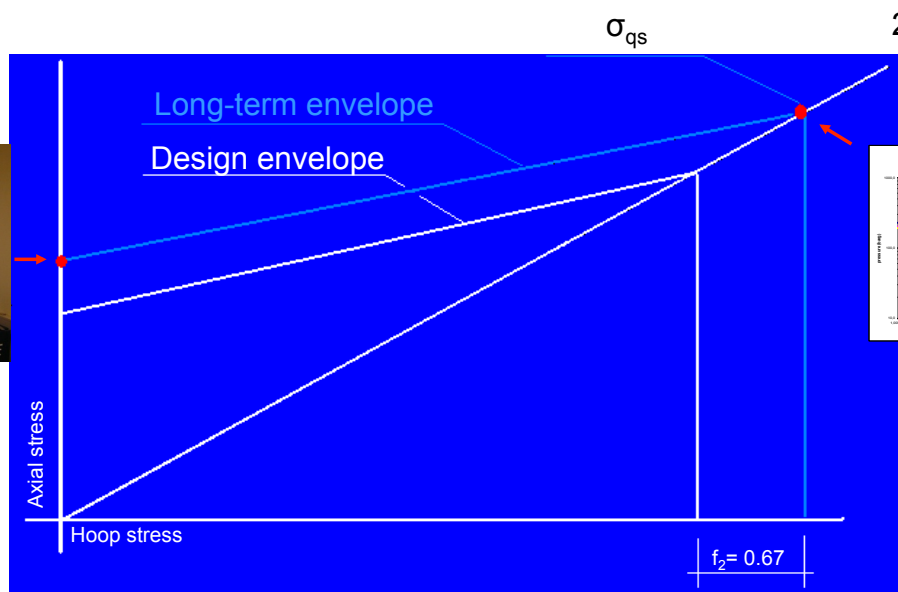
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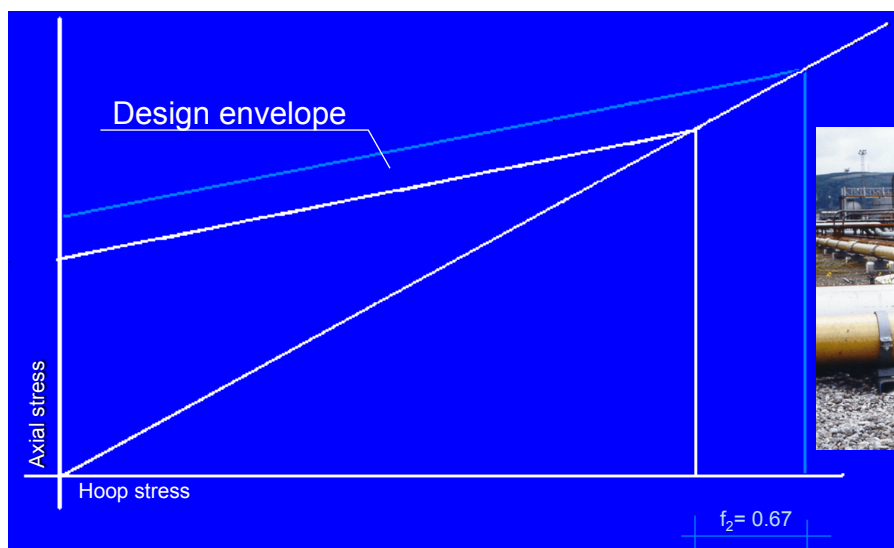
# Several $f_2$ safety factor values to create different design envelopes



Table 3 — Default values for  $f_2$

Loading type	Load duration	$f_2$	Example of loading type
Occasional	Short-term	0,89	Hydrotest
Sustained including thermal loads	Long-term	0,83	Self-mass plus thermal expansion
Sustained excluding thermal loads	Long- term	0,67	Self-mass

# Construct a design envelope to assess pipe stresses against

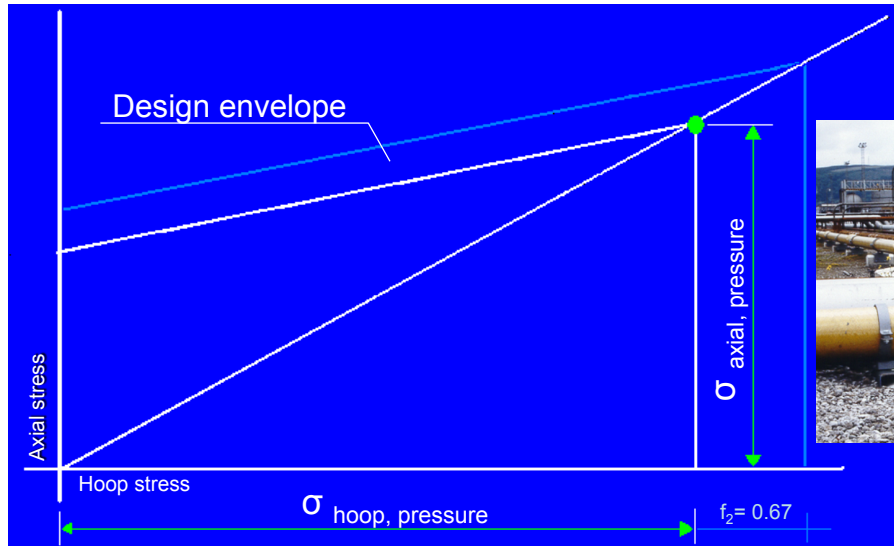




# Construct a design envelope to assess pipe stresses against



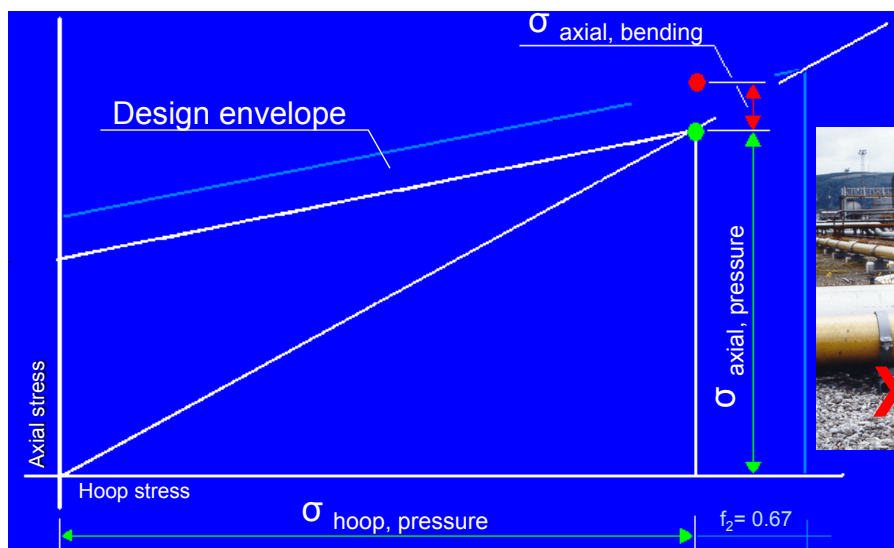
2:1 load condition



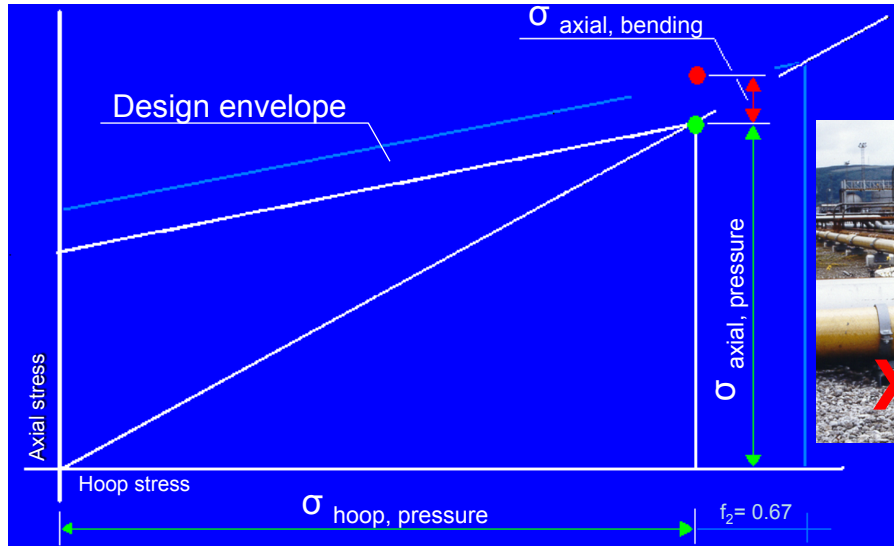
# Construct a design envelope to assess pipe stresses against



2:1 load condition



# Construct a design envelope to assess pipe stresses against



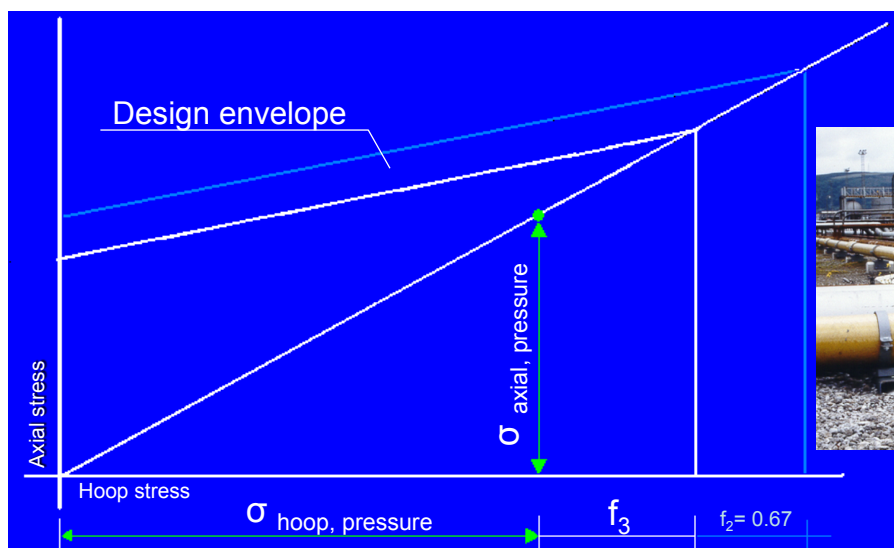
Increase wall thickness!

Internal Pressure

X Bending



# Construct a design envelope to assess pipe stresses against



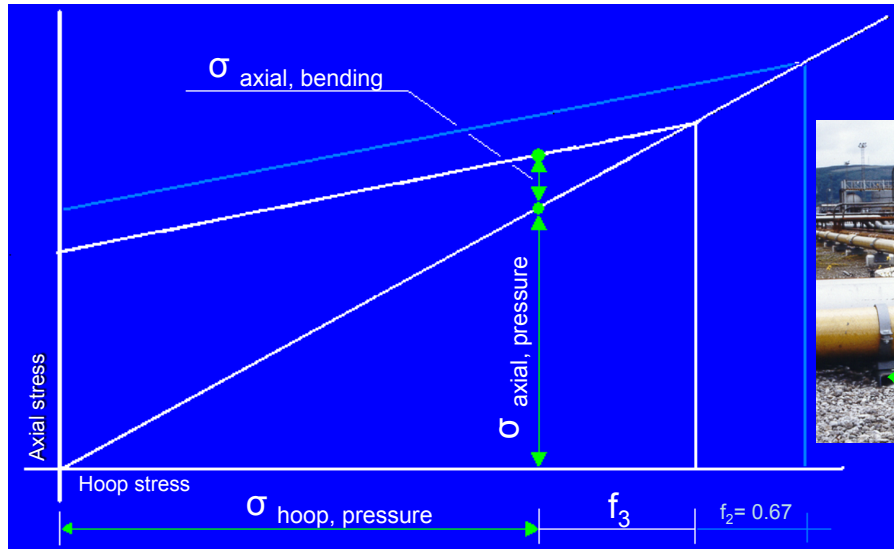
Internal Pressure



# Construct a design envelope to assess pipe stresses against



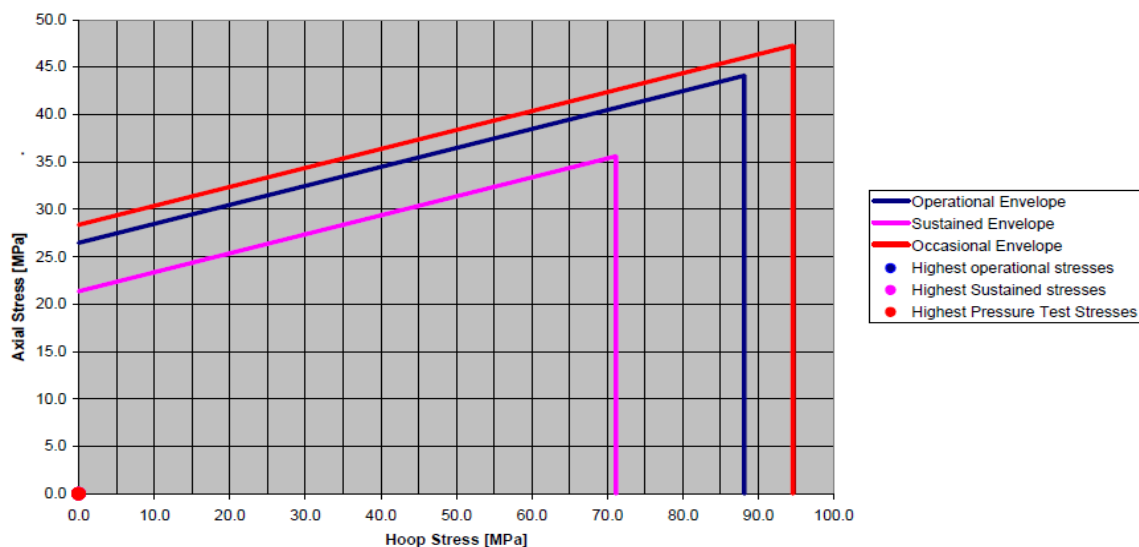
2:1 load condition



## Result: ISO 14692 design envelop that is producer and pipe specific



Stress Assessment & Allowable Stress Envelopes at 65 deg C Conform ISO 14692



# Agenda



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- Using C2

## Configuration Options C2 for FRP



*Typical values from a supplier:*

$E_a = 12,000 \text{ MPa}$

$E_h = 20,400 \text{ MPa}$   $E_h E_a = 1.7$

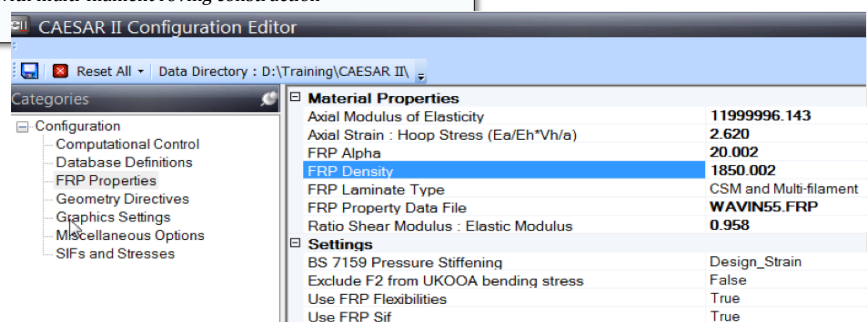
$E_h E_a \nu_{ha} = 0.38$  (Poisson Ratio)

$\text{Density} = 1849 \text{ kg/m}^3$

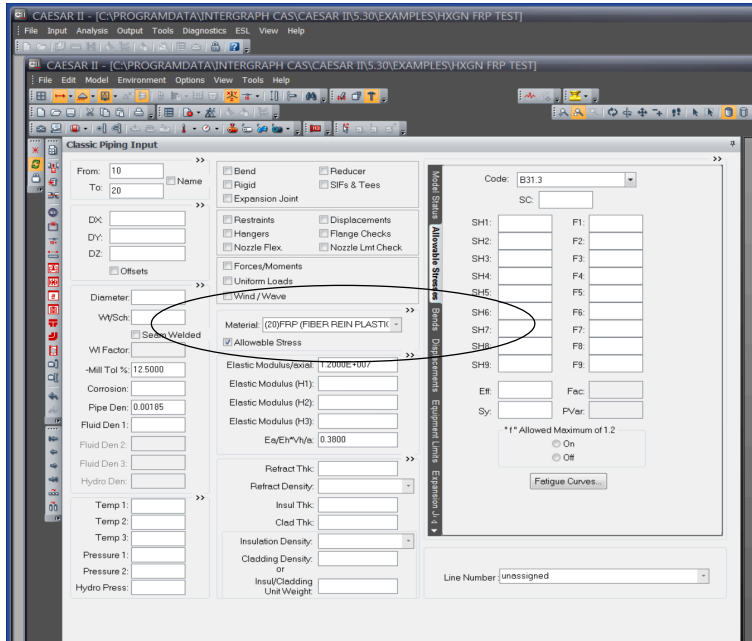
$\text{CTE} = 206 \times 10^{-6} \text{ mmmm}^\circ\text{C}$  (thermal expansion)

$\text{Shear Modulus} = 11,496 \text{ MPa}$

*Bend Laminate Type = Chopped Strand mat with multi-filament roving construction*



# Select FRP material while building your model



Activates Orthotropic material model of C2

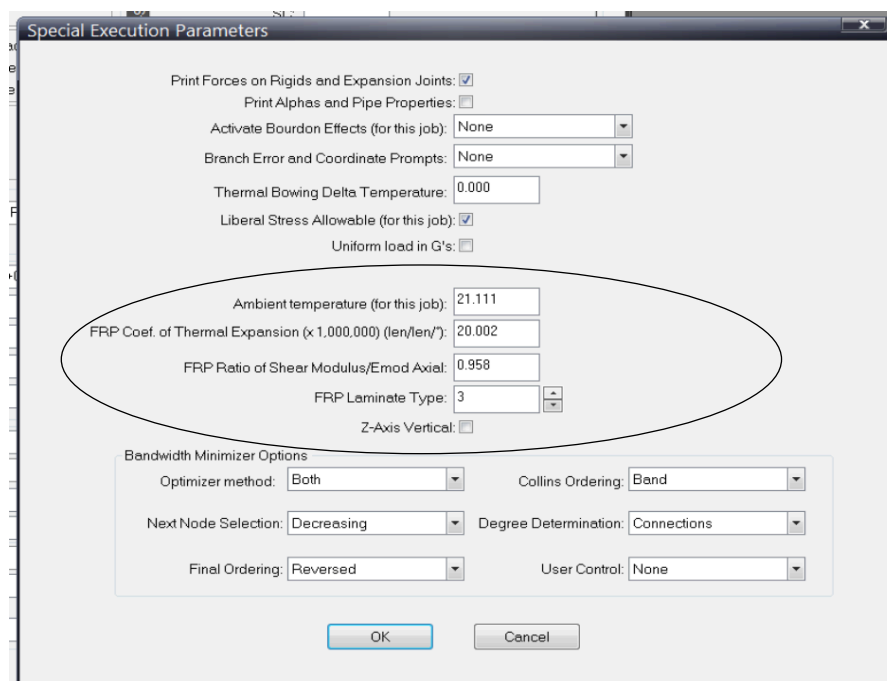
Requires several parameters for axial and hoop direction

- Elastic Modulus
- Poisson Ratio
- Shear modulus

Be careful with diameters and wall thicknesses

Use Fitting data from supplier to ensure the right laminate thickness is entered everywhere

# Check the special execution parameters



# Setting up the Design Envelope using values provided by supplier



Code: ISO 14692

Failure Envelope for Plain Pipe

al(0:1): 61.200 al(2:1): 62.500  
 al(1:1): hl(2:1): 125.000  
 hl(1:1):

Failure Envelope for Joints/Fittings

Joints Bends

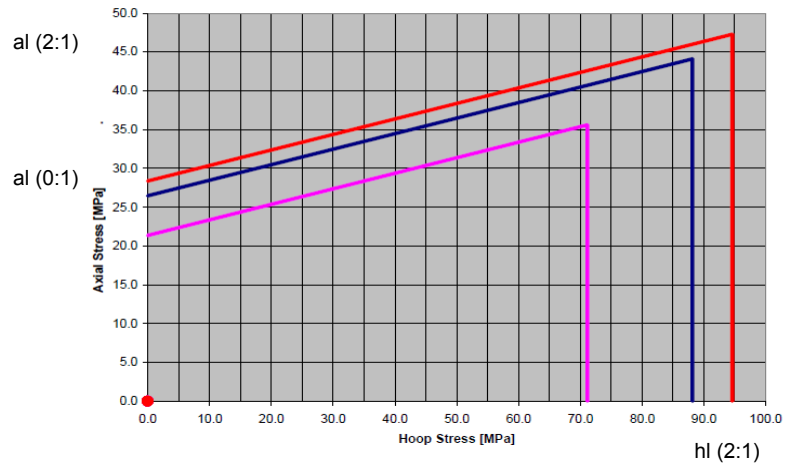
Qs: 75.000 Qs: 75.000  
 r: 2.000 r: 1.000  
 Tees (r=1.0) Eh/Ea: 1.700  
 Qs: 75.000 ☐ Hand Lay

Partial Factors for Temperature (A1)

1: 1.000 4: 7:  
 2: 5: 8:  
 3: 6: 9:

Other Partial Factors

Chemical Resistance (A2): 1.000  
 Cyclic Service (A3): 1.000  
 System Design Factor: 0.670  
 Thermal Factor (k): 0.850



**Run and evaluate the results is similar to steel except the stresses are evaluated per direction**

